

# Thermodynamique

## Changements d'états

### Exercice 1

eau à 100°C	h	u	s	v
V	2676	<del>1,6729</del> 2506,5	7,35	1,6729
L	419	418,94		

$$1) \cdot h_{\text{vap}}(100^\circ) = h_V - h_L = 2257 \text{ kJ/kg}$$

$$\cdot s_{1 \rightarrow 2}(T_0) = \frac{h_{1 \rightarrow 2}(T_0)}{T_0} \quad \text{ici } s_{\text{vap}}(100^\circ\text{C}) = \frac{h_{\text{vap}}(100^\circ\text{C})}{373,15} = 6,05 \text{ kJ} \cdot \text{kg}^{-1} \cdot \text{K}^{-1}$$

$$\text{on } s_{\text{vap}} = s_V - s_L = 1,30 \text{ kJ} \cdot \text{kg}^{-1} \cdot \text{K}^{-1}$$

$$2) u_{\text{vap}} = u_V - u_L = 2087,56 \text{ kJ} \cdot \text{kg}^{-1}$$

$$h_{\text{vap}} = u_{\text{vap}} + p^{\text{sat}}(T)(v_V - v_L)$$

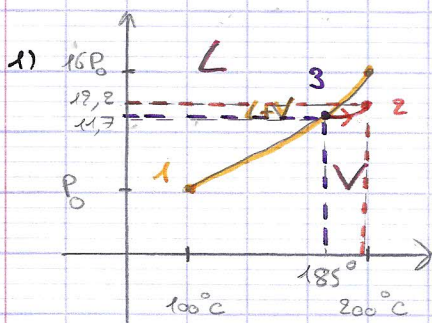
$$v_L = v_V - \frac{h_{\text{vap}} - u_{\text{vap}}}{p^{\text{sat}}(T)} = 0,0011 \text{ m}^3/\text{kg}$$

$$\text{on } \rho_L = \frac{1}{v_L} = 909 \text{ kg/m}^3 \Rightarrow \text{Faux car les mesures}$$

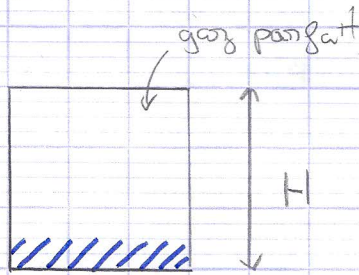
$$\text{moment que } \rho_L(100^\circ, 1 \text{ atm}) = 961 \text{ kg} \cdot \text{m}^{-3}$$



## Exercice 3: Eau liquide et eau vapeur dans une chaudière



2)



$$PV = m_v R (\theta_1 + 273)$$

$$V = (H - h)S$$

$$V \sim HS \text{ car } h \ll H$$

équation liq-vap,  $\theta = \theta_1 = 100^\circ\text{C}$

$$\Rightarrow P = P^s(\theta_1) = P_0$$

$$m_{v,1} = \frac{P_0 SH}{R(\theta_1 + 273)}$$

$$= 32,1 \text{ mol}$$

3)  $m_{p,1} = ?$

$$M = 18 \text{ g} \cdot \text{mol}^{-1}$$

$$\rho = 1 \text{ g} \cdot \text{cm}^{-3}$$

$$m = \rho V = \rho Sh$$

$$m_{p,1} = \frac{m}{M} = \frac{\rho Sh}{M}$$

$$m_{p,1} = 277,8 \text{ mol}$$

$$4) \quad m_{v,1} = \frac{P^s(\theta_1) SH}{R(\theta_1 + 273)}$$

$$= \frac{\left(\frac{\theta_1}{100}\right)^4}{(\theta_1 + 273)} \cdot \frac{P_0 SH}{R}$$

$m_v$  devient donc

$$m_v \approx \frac{P_0 SH}{R} \cdot \frac{(\theta/100)^4}{\theta + 273}$$

5) état 2,  $\theta_2 = 200^\circ\text{C}$ ,  $m_{liq,2} = 0$

$$\Rightarrow m_{vap,2} = m_{tot} = m_{vap,1} + m_{p,1}$$

$$P_2 V = m_{tot} R (\theta_2 + 273)$$

$$\Rightarrow P_2 = \frac{(m_{vap,1} + m_{p,1}) R (\theta_2 + 273)}{SH}$$

$$= 12,2 \text{ bars}$$

6) Dernière goutte de liquide ③

$$m_{p,3} = 0 \quad \text{mais} \quad P_3 = P^{sat}(\theta_3)$$

$$m_{vap,3} = 0$$

$$\Rightarrow \text{que du gaz} \Rightarrow P_3 V = m_{tot} R (\theta_3 + 273)$$

$$P_0 \left(\frac{\theta_3}{100}\right)^4 SH = (m_{p,1} + m_{vap,1}) R (\theta_3 + 273)$$

On a donc un polynôme de la forme  $a\theta^4 + b\theta + c = 0$

$$m_{\text{tot}} = m_V = \frac{P_0 S H}{R} \frac{(0/100)^4}{\theta + 273}$$

$$\Rightarrow \theta_3 = 185^\circ \text{C}$$

$$P^s(\theta_3) = 11,7 \text{ bars}$$

### Exercice 4: détente d'une vapeur saturante sèche

détente :  $V \nearrow$

$$T_1 \rightarrow T_2$$

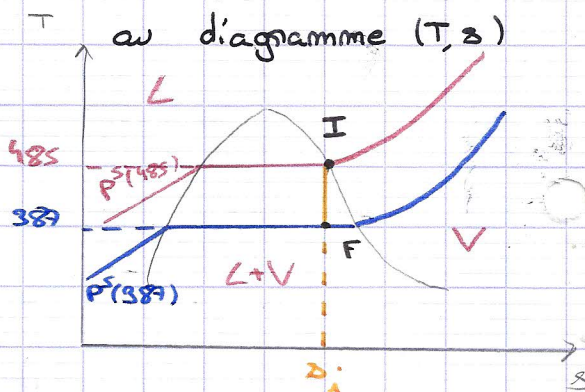
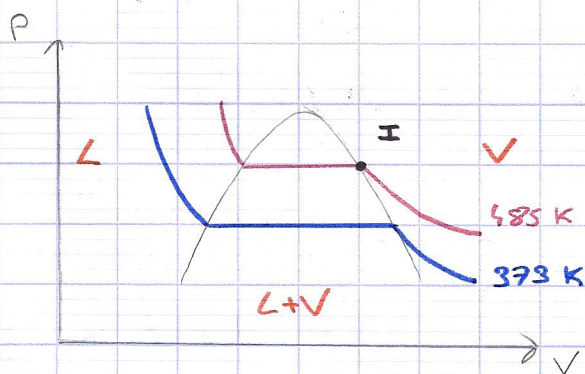
$$185 \text{K} \rightarrow 373 \text{K}$$

vapeur saturante sèche  $\rightarrow x=1$

$\hookrightarrow$  limite du liquide

on passe donc

au diagramme  $(T, s)$



$$v_i = \frac{V_i}{m}$$

La transformation est une isentropie

$$v_g = \frac{V_g}{m}$$

$\Rightarrow s = \text{cte}$

$x_i = 1$  on a 100% de vapeur

$$x = \frac{LM}{LV} = \frac{s(M) - s(L)}{s(V) - s(L)} = 0,83 \quad \text{donc } 83\% \text{ de vapeur}$$

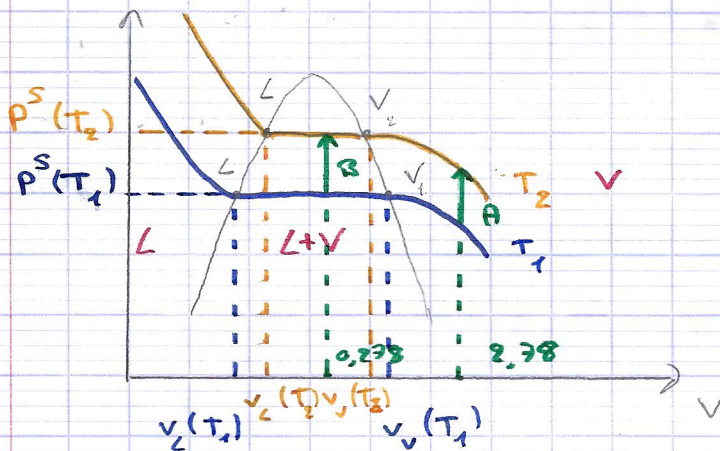
$$x = \frac{LM}{LV} = \frac{v(M) - v_L}{v_V - v_L}$$

$$v(M) = x v_V + (1-x) v_L$$

$$\Rightarrow v_g = 1,416 \text{ m}^3 \cdot \text{kg}^{-1}$$

## Exercice 6: Chauffage d'une marmite

1)



2)

$$v_A = \frac{V_A}{m_A} = 2,78 \text{ m}^3 \cdot \text{kg}^{-1} \quad x_A = 1 \quad \text{vapeur}$$

$$v_B = \frac{V_B}{m_B} = 0,278 \text{ m}^3 \cdot \text{kg}^{-1} \quad x_B = \frac{L_B}{L_V} = 16 \%$$

Transformation  $T_A \rightarrow T_B$

3) a)

$v = \text{cte}$  schéma  $\Rightarrow$  A' vapeur  
B' liq + vap

$$x_B' = \frac{v - v_L(T_2)}{v_V(T_2) - v_L(T_2)} = 0,71\%$$

b)



c) On cherche Q :

$$Q = \Delta U$$

$$= \Delta U_A + \Delta U_B$$

$$\begin{aligned} \bullet \underline{A \rightarrow A'} : \text{vapeur} \rightarrow \text{g.p} &\rightarrow \Delta U_{A'A} = C_V (T_2 - T_1) \\ &= m_A C_V (T_2 - T_1) \end{aligned}$$

$$\begin{aligned} \bullet \underline{B \rightarrow B'} \quad \Delta U_{B'B} &= \Delta U_{BV_1} + \Delta U_{V_1V_2} + \Delta U_{V_2B'} \\ &= m_B C_V (T_2 - T_1) \end{aligned}$$

$$\begin{aligned} \Delta U_{BV_1} &= m_B \Delta U_{BV_1} \\ &= m_B \int_B^{V_1} du \end{aligned}$$

$$dh = du + p^{sat} dv$$

$$\Rightarrow du = dh - p^{sat}(T) dv$$

$$\Rightarrow \Delta U_{BV_1} = \int_B^{V_1} dh - \int_B^{V_1} p^{sat}(T_1) dv$$

$$\begin{aligned} \text{on } dh &= h_{1 \rightarrow 2}(T) dx_2 \\ &= h_{vap}(T) dx \end{aligned}$$

$$\Delta U_{BV_1} = \int_{x_B}^1 h_{vap}(T_1) dx - p^{sat}(T_1) \int_B^{V_1} dv$$

$$\Delta U_{BV_1} = h_{vap}(T_1) [1 - x_B] - p^{sat}(T_1) [v_V(T_1) - v(B)]$$

$$\Delta U_{V_2B'} = h_{vap}(T_2) [x_{B'} - 1] - p^{sat}(T_2) [v(B') - v_V(T_2)]$$

donc :

$$\begin{aligned} \Delta U &= m_A C_V (T_2 - T_1) + m_B C_V (T_2 - T_1) + m_2 h_{vap}(T_1) [1 - x_B] \\ &+ m_B h_{vap}(T_1) [x_{B'} - 1] - m_B p^{sat}(T_1) [v_V(T_1) - v_B] \\ &- m_B p^{sat}(T_2) [v(B') - v_V(T_2)] \end{aligned}$$

$$\underline{d)} \quad \Delta S = \Delta S_{A \rightarrow A'} + \Delta S_{B \rightarrow B'} \quad \Rightarrow \quad dS = c_v \frac{dT}{T} + mR \frac{dV}{V}$$

$$du = Tds - PdV$$

$$dS = \frac{1}{T} du + \frac{P}{T} dV$$

$$\Delta S = \int_A^{A'} dS = \int_{T_1}^{T_2} c_v \frac{dT}{T} + \int_{V_1}^{V_2} mR \frac{dV}{V}$$

on passe par

$$\begin{cases} du = c_v dT \\ \frac{P}{T} = \frac{mR}{V} \end{cases}$$

$$\Delta S_{AA'} = c_v \ln \left( \frac{T_2}{T_1} \right)$$

$$\Delta S_{AA'} = m_A c_v \ln \left( \frac{T_2}{T_1} \right)$$

B → B' en passant par  $L_1$  et  $L_2$

$$\Delta S_{BB'} = \Delta S_{BL_1} + \Delta S_{L_1 L_2} + \Delta S_{L_2 B'}$$

$$\Delta S_{L_1 L_2} = g_{ac, lq}$$

$$dh = c dt$$

$$ds = \frac{dh}{T} = c \frac{dT}{T}$$

$$\Delta S_{L_1 L_2} = m_B \cdot c \cdot \ln \left( \frac{T_2}{T_1} \right)$$

$$ds_{1 \rightarrow 2} = \frac{dh_{1 \rightarrow 2}}{T_0} \quad \text{donc} \quad \Delta S_{BL_1} = \int_B^{L_1} \frac{h_{vap}(T_1)}{T_1} dx$$

$$\Rightarrow \Delta S_{BL_1} = \frac{h_{vap}(T_1)}{T_1} (-x_B)$$

$$\Delta S_{BL_2} = \frac{h_{vap}(T_2)}{T_2} (x_{B'})$$

Bilan

$$\Delta S = m_A \cdot c_v \ln \left( \frac{T_2}{T_1} \right) + m_B c_{liq} \ln \left( \frac{T_2}{T_1} \right)$$

$$+ m_B \left[ \frac{h_{vap}(T_2) x_{B'}}{T_2} - \frac{h_{vap}(T_1) x_B}{T_1} \right]$$

$$= 5535,65 \text{ J.K}^{-1}$$

2<sup>nd</sup> principe:

$$dS_{\text{source}} = \frac{\delta Q_{\text{source}}}{T_{\text{source}}} = -\frac{\delta Q}{T_{\text{source}}}$$

ici  $T_{\text{source}} = T_2$

$$\Rightarrow \Delta S_{\text{source}} = - \int \frac{\delta Q}{T_2} = - \frac{1}{T_2} \int dq$$

$$\Delta S_{\text{source}} = - \frac{Q}{T_2} = - 5212 \text{ J} \cdot \text{K}^{-1}$$

ou 2<sup>nd</sup> principe  $\Delta S = \Delta S_{\text{ech}} + \Delta S_{\text{créé}}$

$$\Rightarrow \Delta S = \frac{Q}{T_2} + S_{\text{créé}}$$

Donc  $S_{\text{créé}} = \Delta S - \frac{Q}{T_2} = 323 \text{ J} \cdot \text{K}^{-1}$

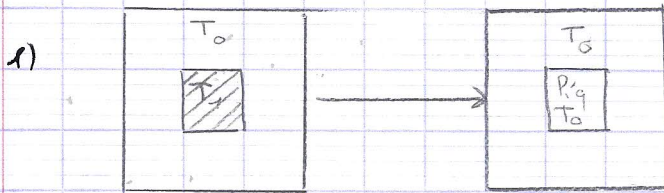
cas du système A:

$$S_{\text{ech}} = \frac{Q}{T_2} = \frac{m c_V (T_2 - T_1)}{T_2} = m c_V \left(1 - \frac{T_1}{T_2}\right)$$

$$\Rightarrow \Delta S_A = m c_V \ln \left(\frac{T_2}{T_1}\right)$$



## Exercice 2: Transfert thermique et état métastable



$$\Delta H = \Delta H_1 + \Delta H_2$$

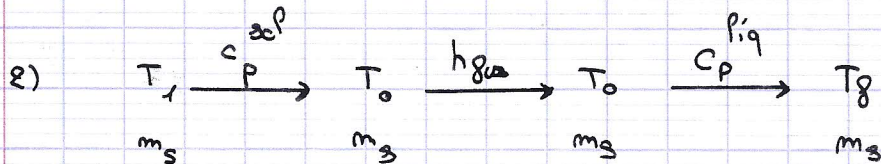
$\Delta H_1 = ?$  corps solide  $dh = c dT$   
 $\Rightarrow \Delta H = m \Delta h = m \int_1^0 dh$   
 $= m \int_1^0 c dT$   
 $\Delta H_1 = m_s c_{p,s} (T_0 - T_1)$

Changement d'état

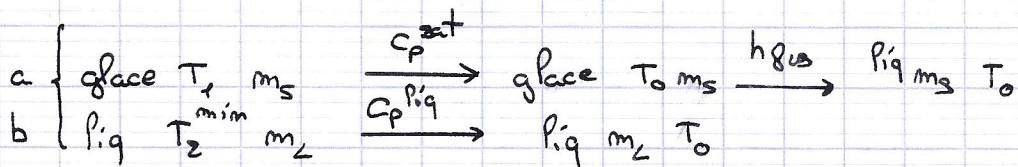
$$\Delta H_2 = m_s h_{\text{fus}}(T_0)$$

$$\Delta H = m_s c_{p,s} (T_1 - T_0) + m_s h_{\text{fus}}(T_0)$$

$$\Delta H = 375,8 \text{ kJ}$$



ici  $T_2$  min tel que  $T_g = T_0$  et glace fondue



$$\Delta H_a = m_s c_{p,s} (T_0 - T_1) + m_s h_{\text{fus}}$$

$$\Delta H_b = m_2 c_p^{\text{liq}} (T_0 - T_2^{\text{min}})$$

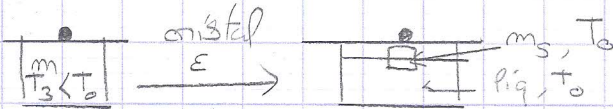
ici calorimètre isolé et capacité calorifique = 0

$$\Delta H_{\text{tot}} = 0$$

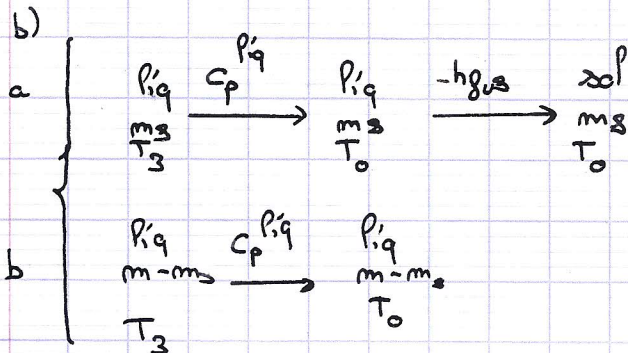
$$\Rightarrow \Delta H_a + \Delta H_b = 0 \Rightarrow T_2^{mim} = T_0 + \frac{m_s c_p^{sol} (T_0 - T_1) + m_s h_{fus}}{m_c c_p^{liq}}$$

3) m

a)  $T_3 < T_0$



$T_{finale} = T_0$   
équilibre liq + solide



$$\Delta H = \Delta H_a + \Delta H_b$$

$$\Delta H_a = c_p^{liq} m_s (T_0 - T_3) - m_s h_{fus}$$

$$\Delta H_b = (m - m_s) c_p^{liq} (T_0 - T_3)$$

$$\Rightarrow \Delta H = 0$$

$$\Rightarrow m c_p^{liq} (T_0 - T_3) - m_s h_{fus} = 0$$

$$m_s = \frac{m c_p^{liq} (T_0 - T_3)}{h_{fus}}$$

c)  $\Delta S^{réelle} = S_{ech}^{réelle} + S_{cncé}^{réelle}$

j'imagine une transformation réversible ayant les mêmes états i et f.

$$\Rightarrow \Delta S^{img} = S_{ech}^{img} + \cancel{S_{cncé}^{img}} = \Delta S^{réelle}$$

6

→  $\Delta S_{\text{réelle}} = \Delta S_{\text{calcule}} = \Delta S_{\text{éch}}^{\text{img}} = \Delta S_{\text{éch}}^{\text{img}}$

→  $\Delta S = \Delta S_e^{\text{img}} + \Delta S_b^{\text{img}}$

$\Delta S_b^{\text{img}} = ?$  modèle liquide  $ds = c \frac{dT}{T}$

→  $\Delta S_b = (m - m_s) \int_{T_3}^{T_0} c \frac{dT}{T}$

$\Delta S_b = (m - m_s) c \ln \left( \frac{T_0}{T_3} \right)$

$\Delta S_a^{\text{img}} = ?$

partie 1 pour liq  $T_3 \rightarrow T_0$

$\Delta S_a^1 = m_s c \ln \left( \frac{T_0}{T_3} \right)$

partie 2 "chgmt d'état"

$ds_{p \rightarrow s} = \frac{dh_{p \rightarrow s}}{T_0}$

$\Delta S_{p \rightarrow s} = \frac{\Delta h_{p \rightarrow s}}{T_0}$

⇒  $\Delta S_a = \frac{\Delta h_{p \rightarrow s}}{T_0} = \frac{-m_s h_{fs}}{T_0}$

Bilan

$\Delta S^{\text{img}} = (m - m_s) c \ln \frac{T_0}{T_3} + m_s c \ln \frac{T_0}{T_3} - \frac{m_s h_{fs}}{T_0}$

$\Delta S^{\text{img}} = m c_p^{\text{liq}} \ln \left( \frac{T_0}{T_3} \right) - \frac{m_s h_{fs}}{T_0}$

d)  $m_s = 0,125 \text{ kg}$

$\Delta S^{\text{néel}} = \Delta S^{\text{img}} = 3,05 \text{ J.K}^{-1}$

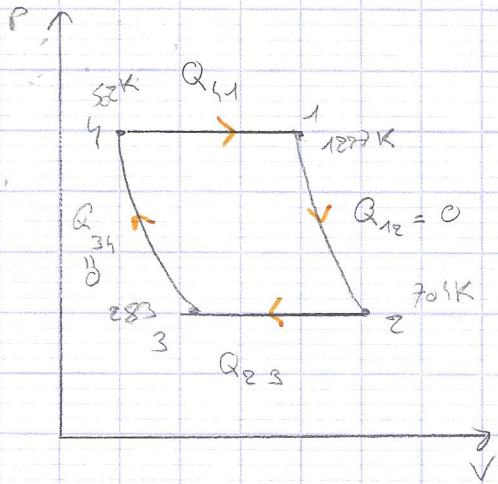
$= S_{\text{éch}}^{\text{néel}} + S_{\text{créé}}^{\text{néel}}$

⇒  $S_{\text{créé}}^{\text{néel}} = 3,05 \text{ J.K}^{-1}$  donc irréversible

||  
O car système isolé

exercice 3

1)



Izentrópique  $P \propto \frac{1}{V^\gamma}$

	T	P
1	1223	7
2	?	1
3	283	1
4	?	7

2)  $e = -\frac{W}{Q_c}$

$\Delta U = W + Q = 0$   
 $\Rightarrow W + Q_c + Q_g = 0$

$e = 1 + \frac{Q_g}{Q_c}$

3) 1 → 2 adiabatique réversible = izentrópique

$$\begin{cases} ds = \delta S_{ech} + \delta S_{cnc\acute{e}} \\ ds = \frac{\delta Q}{T} + \delta S_{cnc\acute{e}} \end{cases}$$

adiabatique  $\Rightarrow \delta Q = 0$   
 $\Rightarrow Q_{12} = 0$

4)  $\begin{matrix} 2 \rightarrow 3 \\ 1 \rightarrow 1 \end{matrix} \left\{ \begin{matrix} \text{réversible} \\ \text{+ isobare} \end{matrix} \right.$

$dH = Tds + VdP$  isobare  $dP = 0$   $dH = Tds$

2<sup>ème</sup> principe :

$$\begin{aligned} ds &= \delta S_{ech} + \delta S_{cnc\acute{e}} \\ &= \frac{\delta Q}{T} + \delta S_{cnc\acute{e}} \end{aligned}$$

$\Rightarrow Tds = \delta Q$

$$s) dH = \delta Q \quad \text{sur } 2 \rightarrow 3 \quad \Rightarrow \int_2^3 \delta Q = \int_2^3 dH$$

$$1 \rightarrow 1$$

$$Q_{23} = \Delta H_{23}$$

gaz parfait  $\Rightarrow \Delta H_{23} = C_p (T_3 - T_2)$

$$Q_{23} = C_p (T_3 - T_2)$$

idem  $Q_{11} = C_p (T_1 - T_1) > 0$

6)  $1 \rightarrow 2$  Loi de Laplace  
 $3 \rightarrow 1$

$$T^\gamma P^{1-\gamma} = cte$$

$$\frac{T_1^\gamma P_1^{1-\gamma}}{T_2^\gamma P_2^{1-\gamma}} = 1$$

$$\Rightarrow T_2 = T_1 \left( \frac{P_2}{P_1} \right)^{\frac{1-\gamma}{\gamma}}$$

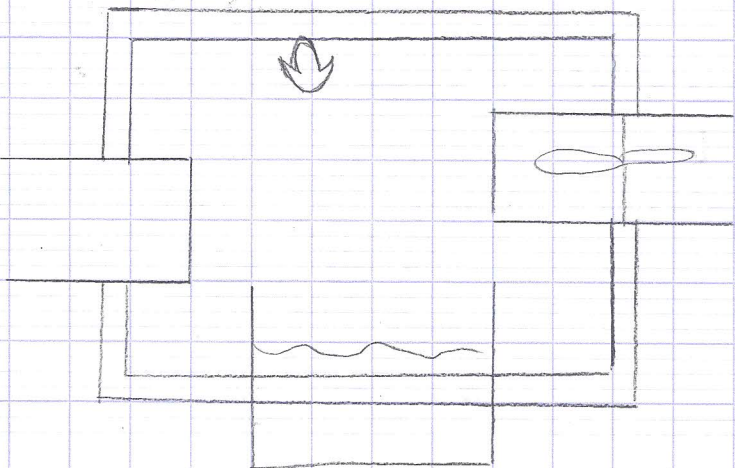
$$T_2 = 1227 \left( \frac{1}{7} \right)^{\frac{0.4}{1.4}} = 704 \text{ K}$$

idem  $T_1 = T_3 \left( \frac{P_1}{P_3} \right)^{\frac{1-\gamma}{\gamma}} \approx 502 \text{ K}$

7)  $Q_c > 0$  donc  $Q_c = Q_{11} (T_1 > T_1)$   
 $Q_g < 0$  donc  $Q_g = Q_{23} (T_3 < T_2)$

$$\text{on } e = 1 + \frac{Q_g}{Q_c} = 1 - \frac{C_p (T_3 - T_2)}{C_p (T_1 - T_1)} \Rightarrow e = 1 - \frac{T_2 - T_3}{T_1 - T_1}$$

8)



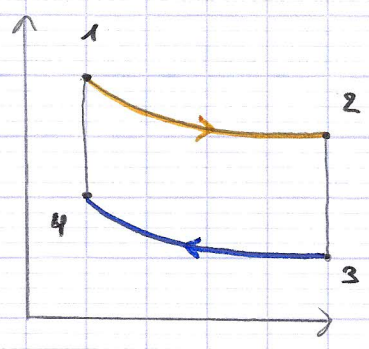
Exercice 10

1) Imcompos

$P_1, P_2$

$$PV = nRT$$

$$\Rightarrow n = 3,76 \cdot 10^{-2} \text{ mol}$$



$$W_{12} = -1,08 \text{ kJ} = -nRT_c \ln\left(\frac{V^+}{V^-}\right)$$

$$Q_{12} = 1,08 \text{ kJ} = -W_{12}$$

$$Q_c > 0$$

$$W_{23} = 0$$

$$Q_{23} = \Delta U_{23} = \frac{nR}{\gamma-1} (T_2 - T_c) = -0,922 \text{ kJ}$$

$$W_{34} = nRT_2 \ln\left(\frac{V^+}{V^-}\right) = 0,23 \text{ kJ}$$

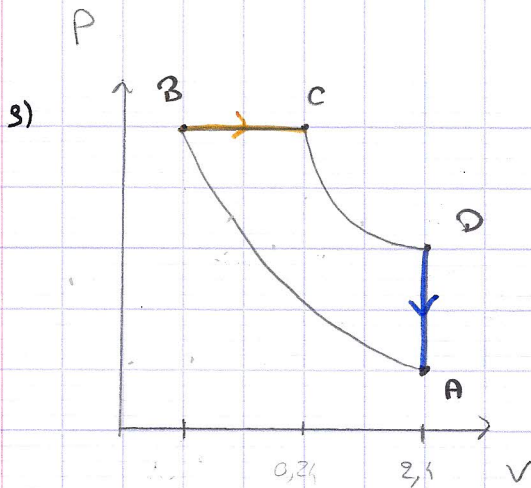
$$Q_{34} = -W_{34} = -0,23 \text{ kJ}$$

$$e = \frac{-W}{Q_c} = 1 + \frac{Q_3}{Q_c} = 0,79 = e_c \quad \text{\AA} \quad \text{cause de n\u00e9g\u00e9-}$$

menaten.

Exercice 11

1)		A	B	C	D
ben	P	1	44,3	4,3	1,76
L	V	2,1	0,16	0,24	2,1
K	T	323	954	1431	570



matériau 2

$$TV^{\gamma-1} = \text{cte}$$

$$V_B = V_A \left( \frac{T_A}{T_B} \right)^{\frac{1}{\gamma-1}}$$

$$= 0,16 \text{ L}$$

$$PV = nRT \Rightarrow P_B = 44,3 \text{ bars}$$

$$PV = nRT \Rightarrow T_C = 1131 \text{ K}$$

$$TV^{\gamma-1} = \text{cte} \text{ sur CD}$$

$$\Rightarrow T_D = 570 \text{ K}$$

$$P_D = 1,76 \text{ bars}$$

$$2) C_V = \frac{nR}{\gamma-1} = 1,86 \text{ J.K}^{-1}$$

$$C_P = \gamma C_V = 2,6 \text{ J.K}^{-1}$$

1) A  $\rightarrow$  B sont fermées

$$\Delta U_{AB} = W_{AB} + Q_{AB}$$

$\rightarrow$  adiabatique réversible  $Q_{AB} = 0$

$$\text{donc } W_{AB} = \Delta U_{AB}$$

$$\text{on } \Delta U_{AB} = C_V (T_B - T_A) \quad \text{car gaz parfait}$$

$$W_{AB} = C_V (T_B - T_A) = 1170 \text{ J}$$

$$3) \Delta U_{BC} = C_V (T_C - T_B) = W_{BC} + Q_{BC}$$

$$\Delta H_{BC} = C_P (T_C - T_B) = Q_{BC}$$

$$dU = Tds - PdV = \delta W + \delta Q$$

$$dH = Tds + VdP$$

$$dH = \delta W + \delta Q + PdV + VdP$$

$$Q_{BC} = C_p (T_c - T_B)$$

$$W_{BC} = C_v (T_c - T_B) - C_p (T_c - T_B)$$

$$6) \begin{cases} W_{CO} = ? \text{ idem} \\ Q_{CO} = ? \\ Q_{OC} = 0 \\ W_{CO} = C_v (T_D - T_c) \end{cases}$$

$$7) \text{ isochore } \begin{cases} W_{DA} = 0 \\ \Delta U_{DA} = Q_{DA} \end{cases} \text{ donc } Q_{DA} = C_v (T_A - T_D)$$

$$8) W_{ABCD} = C_v (T_B - T_A) + C_v (T_c - T_B) - C_p (T_c - T_B) + C_v (T_D - T_c) + 0$$

$$= C_v (T_D - T_A) - C_p (T_c - T_B)$$

$$= -780 \text{ J}$$

$$9) e = \frac{u}{d} = \frac{-W}{Q_c} = \frac{-W_{ABCD}}{Q_{BC}}$$

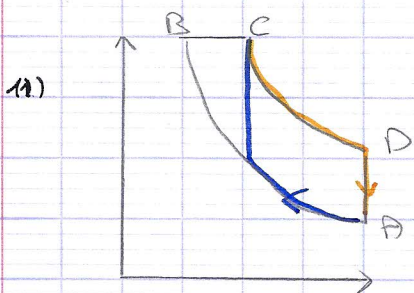
$$= 1 - \frac{C_v}{C_p} \left( \frac{T_D - T_A}{T_c - T_B} \right)$$

$$= 1 - \frac{1}{\gamma} \frac{T_D - T_A}{T_c - T_B} = 0,63$$

$$10) e_{\text{max}} = e_{\text{Carnot}} = 1 - \frac{T_B}{T_c}$$

$$= 1 - \frac{T_A}{T_c} = 0,77$$

dimensionnelle



$$12) \begin{cases} W_{AB_1} = \\ Q_{AB_1} = \\ \text{if faut } T_{B_1} \\ P_A V_A = P_{B_1} V_{B_1} \\ \Rightarrow P_{B_1} = 25,11 \text{ bar} \\ T_{B_1} = 811 \text{ K} \end{cases}$$

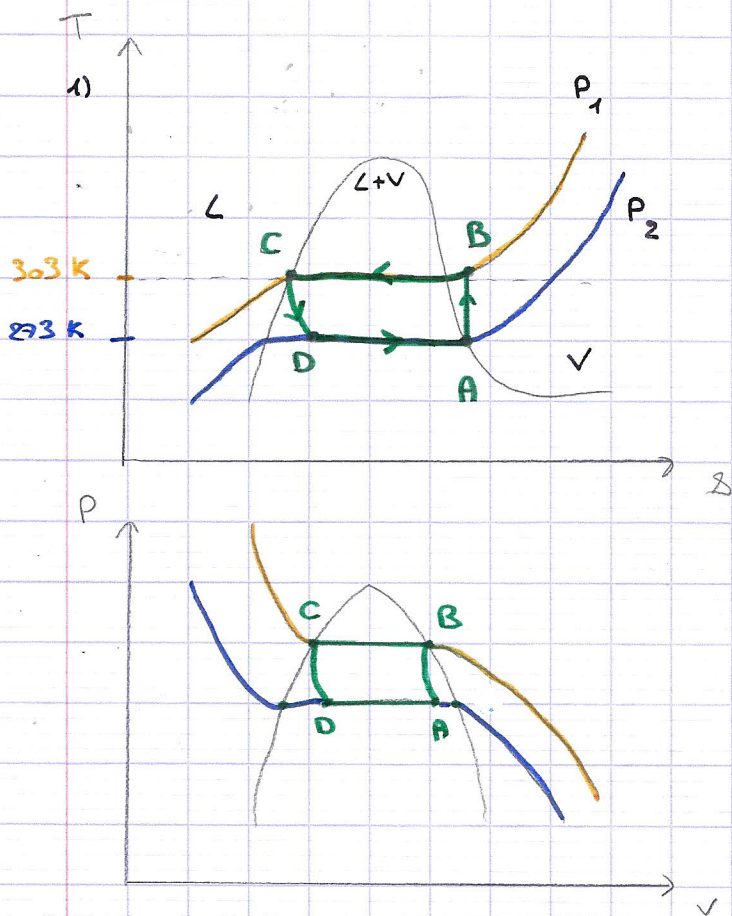
$$\Rightarrow \begin{cases} W_{AB_1} = C_v (T_{B_1} - T_A) \\ Q_{AB_1} = 0 \end{cases}$$

$$\Rightarrow \begin{cases} W_{BC} = 0 \\ Q_{B_1C} = C_v (T_c - T_{B_1}) \end{cases}$$

$$13) e_{BDR} = 1 - \frac{T_D - T_A}{T_c - T_{B_1}} = 0,6$$



## Exercice 12



$$2) \quad \begin{cases} h_C = h_L(303\text{K}) \\ h_D = x_D h_V(273\text{K}) + (1-x_D) h_L(273\text{K}) \end{cases}$$

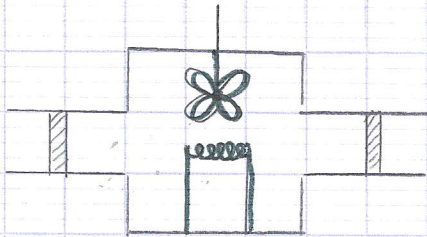
Transfère imaginaire

$$\begin{aligned} \Delta h_{CD} &= \Delta h_{CC'} + \Delta h_{C'D} = 0 \\ &= c(T_D - T_C) + h_{\text{vap}}(273\text{K})(x_D - 0) = 0 \end{aligned}$$

$$x_D = \frac{c(T_C - T_D)}{h_{\text{vap}}(273\text{K})} = 9,7\%$$

3) 1<sup>er</sup> principe adapté au système ouvert

$$\Delta h = w_{\text{autres}} + q$$



Compresseur calorifuge  $\Rightarrow q = 0$

donc  $w_{cp} = w_{autres} = \Delta h_{AB} = h_B - h_A$

gaz parfait  $\Rightarrow \Delta h_{AB} = c_p (T_B - T_A)$

$\Rightarrow w_{cp} = c_p (T_B - T_A)$

$\Rightarrow W = m w_{cp} = m c_p (T_B - T_A)$

1)  $e = \frac{u}{d} = \frac{q_{DA}}{w} = \frac{q_{DA}}{w_{cp}}$

$e = \frac{h_{vap}(273K)(1-x_0)}{c_p (T_B - T_A)}$

$q_{DA} = h_{vap}(273) [1 - x_0]$

$w_{cp} = c_p (T_B - T_A)$

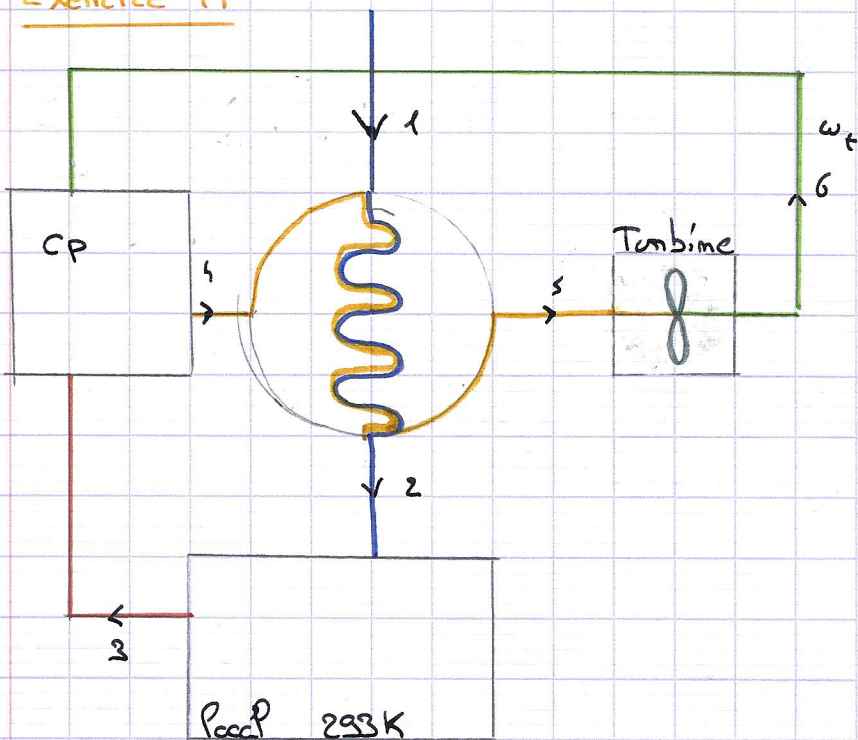
A  $\rightarrow$  B isentropique  $PV^\gamma = cte \Rightarrow P^{1-\gamma} T^\gamma = cte$

$T_B = T_A \left( \frac{P_A}{P_B} \right)^{\frac{1-\gamma}{\gamma}} = T_A \left( \frac{P_B}{P_A} \right)^{\frac{\gamma-1}{\gamma}}$

$e = \frac{h_{vap}(273K)(1-x_B)}{c_p T_A \left[ \left( \frac{P_B}{P_A} \right)^{\frac{\gamma-1}{\gamma}} - 1 \right]} = 7,5$

$e_c = \frac{T_F}{T_C - T_F} = 18,5$

Exercice 11



1) 1 bar  
-5°C

2)  $T_2 = 351 \text{ K}$   
 $P_2 = 1 \text{ bar}$

3)

4) 2 bars  
135 K

5) 2 bars  
351 K

6) 1 bar  
236 K

1)  $3 \rightarrow 1$  }  $isoS \rightarrow T^\gamma P^{1/\gamma} = cte$   
 $5 \rightarrow 6$  }

2)  $w_{35} = w_{cp} = \frac{\gamma R}{(\gamma-1)M} (T_1 - T_3)$

3)  $w_{36} = \frac{\gamma R}{(\gamma-1)M} (T_6 - T_5)$

	1	2	3	4	5	6
P	1	1	1	4	4	1
T	268	351	253	435	351	236

$$2) w_{34} = \frac{\gamma R}{(\gamma-1)M} (T_4 - T_3) = 152,5 \text{ kJ.kg}^{-1}$$

$$3) w_{56} = \frac{\gamma R}{(\gamma-1)M} (T_6 - T_5) = -115,1 \text{ kJ.kg}^{-1}$$

$$4) w_{tot} = w_{56} - w_{34} = 27,1 \text{ kJ.kg}^{-1}$$

$$5) q_c = q_{32} = \Delta h = \frac{\gamma R}{(\gamma-1)M} (T_3 - T_2) = -58,2 \text{ kJ.kg}^{-1}$$

$$6) e_{PRC} = \frac{w}{q} = \frac{-q_c}{w_{56} + w_{34}} = 2,2$$

$$7) e_c = \frac{T_c}{T_c - T_F} = 11,7$$

## Physique de la diffusion

### Exercice 16 : Mesure d'une conductivité thermique.

$$1) \vec{j}_{th} = -\lambda \vec{\nabla} T \quad \text{et} \quad j_{th}(z) = -\lambda \frac{dT}{dz} e_z$$

$$j_{th} = W \cdot m^{-2}$$

$$\lambda = W \cdot m^{-1} \cdot K^{-1}$$

$$2) \varphi_{th} = \iint \vec{j}_{th}(\varphi) \cdot d\vec{S} = \iint j_{th} dS = j_{th}(z) \pi r^2$$

$$3) P = UI = RI^2 \quad \text{donc } P \text{ grand}$$

$$1) \frac{P}{R} = \varphi_{th}$$

$$\frac{U^2}{R} = j_{th}(z) \pi r^2$$

$$j_{th}(z) = \frac{U^2}{R \pi r^2}$$

4) Bilan des flux pendant dt.

$$dU_{in} = j(z, t) dS dt$$

$$dU_{out} = j(z+dz, t) dS dt$$

$$dU = (j(z, t) - j(z+dz, t)) dS dt$$

$$U(t) = u(z, t) dm = u(z, t) dS dz \rho$$

$$U(t+dt) = u(z, t+dt) dS dz \rho$$

$$dU = (\rho u(z, t+dt) - \rho u(z, t)) dS dz$$

$$\frac{\partial u}{\partial t} = -\frac{\partial j}{\partial z} \quad \text{stationnaire donc} \quad \frac{\partial j}{\partial z} = 0$$

$$j_{th}(z) = cte$$

$$j_{th} = -\lambda \frac{dT}{dz} = cte$$

$$T(x) = ax + b$$

$$a) \vec{j}_{\text{th}}(z) = j_{\text{th}} \vec{e}_z$$

$$j_{\text{th}} = -\lambda \frac{dT}{dz} = -\lambda \frac{T_L - T_0}{L}$$

$$\vec{j}_{\text{th}} = \lambda \frac{T_0 - T_L}{L} \vec{e}_z$$

$$\Rightarrow T(z) = \frac{-j_{\text{th}}}{\lambda} z + T_0$$

$$\text{on } j_{\text{th}} = \frac{L^2}{R_{\text{th}} n^2}$$

$$\text{donc } T(z) = -\frac{L^2}{R_{\text{th}} n^2 \lambda} z + T_0$$

$$= -\frac{T_L - T_0}{L} z + T_0$$

$$T(L) = -\frac{L^2}{R_{\text{th}} n^2 \lambda} L + T_0$$

$$T_0 = T_L + \frac{L^2}{R_{\text{th}} n^2 \lambda} L$$

$$b) U = RI$$

$$\vec{j} = R_{\text{th}} \varphi$$

$$T_0 - T_L$$

$$T(z) = \frac{T_L - T_0}{L} z + T_0$$

$$j = \lambda \frac{T_0 - T_L}{L}$$

$$\varphi = j \cdot S = \frac{\lambda S}{L} (T_0 - T_L)$$

$$T_0 - T_L = \varphi \frac{L}{\lambda S}$$

$$R_{\text{th}} = \frac{T_0 - T_L}{\varphi} = \frac{L}{\lambda n^2 S} = 11,1 \text{ K} \cdot \text{W}^{-1}$$

$$\eta_{\text{th}} = \frac{R_{\text{th}}}{L} = \frac{1}{\lambda n^2 S} = 27,8 \text{ K} \cdot \text{W}^{-1} \cdot \text{m}^{-1}$$

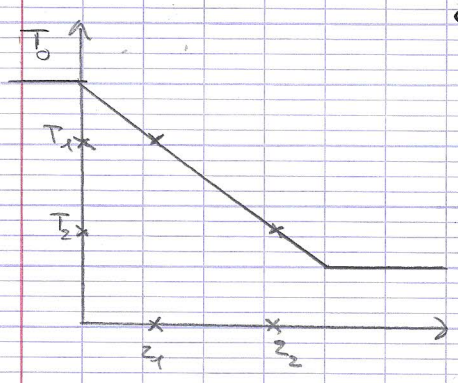
$T(z_1) = T_1$   
 $T(z_2) = T_2$

pente =  $\frac{T_1 - T_2}{z_1 - z_2}$

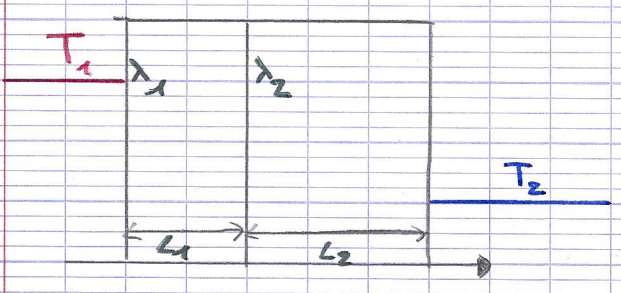
$\frac{j}{\lambda} = \frac{R_{th}^2}{\Delta z} \frac{T_1 - T_2}{z_1 - z_2}$

$j = -\lambda \times \text{pente}$   
 on  $j = \frac{\Delta z^2}{R_{th}^2}$

pente =  $\frac{\Delta z^2}{R_{th}^2}$



Exercice 17: D. Diffusion de chaleur à travers une paroi composite



1) Régime stationnaire  
 $\Delta T(x, y, z) = 0$   
 Problème 1D  $\rightarrow T(x)$

$\frac{d^2 T(x)}{dx^2} = 0$

donc  $\begin{cases} T_1(x) = a_1 x + b_1 \\ T_2(x) = a_2 x + b_2 \end{cases}$  4 inconnues.

2) Continuité de T  
 Continuité de j en stationnaire

4 conditions aux limites

$T_1(0) = T_1$	1
$T_2(L_1 + L_2) = T_2$	2
$T_1(L_1) = T_2(L_1)$	3
$j_1(L_1) = j_2(L_1)$	4

$$\text{on par Fourier} \quad \vec{j}_1 = -\lambda_1 \frac{dT_1}{dx} \quad \vec{j}_1$$

$$\vec{j}_2 = -\lambda_2 \frac{dT_2}{dx} \quad \vec{j}_2$$

$$\Rightarrow \lambda_1 \frac{dT_1}{dx}(L_1) = \lambda_2 \frac{dT_2}{dx}(L_2) \quad |$$

$$1 \Rightarrow T_1(0) = b_1 = T_1 \Rightarrow b_1 = T_1$$

$$2 \Rightarrow T_2(L_1 + L_2) = a_2(L_1 + L_2) + b_2 = T_2$$
$$\underline{a_2(L_1 + L_2) + b_2 = T_2}$$

$$\Rightarrow T_1(x) = \frac{dT_1(x)}{dx} x + T_1 = \underline{a_1 x + T_1}$$

$$T_2(x) = \frac{dT_2}{dx} \cdot x + (T_2 - a_2(L_1 + L_2))$$

$$T_2(x) = \frac{dT_2(x)}{dx} (x - (L_1 + L_2)) + T_2 = \underline{a_2(x - (L_1 + L_2)) + T_2}$$

$$\Rightarrow \lambda_1 a_1 = \lambda_2 a_2 = \alpha$$

$$\text{donc } a_1 = \frac{\alpha}{\lambda_1} \quad a_2 = \frac{\alpha}{\lambda_2}$$

$$T_1(x) = \frac{\alpha}{\lambda_1} x + T_1$$

$$T_2(x) = \frac{\alpha}{\lambda_2} [x - (L_1 + L_2)] + T_2$$

$$3 \Rightarrow T_1(L_1) = T_2(L_1)$$

$$\Rightarrow \frac{\alpha L_1}{\lambda_1} + T_1 = -\frac{\alpha L_2}{\lambda_2} + T_2$$

$$\Rightarrow \alpha \left( \frac{L_1}{\lambda_1} + \frac{L_2}{\lambda_2} \right) = T_2 - T_1$$



$$\Rightarrow \alpha = \frac{T_2 - T_1}{\frac{L_1}{\lambda_1} + \frac{L_2}{\lambda_2}}$$

donc  $T_1(\infty) = \frac{1}{\lambda_1} \left( \frac{T_2 - T_1}{\frac{L_1}{\lambda_1} + \frac{L_2}{\lambda_2}} \right) \alpha + T_1$

$T_2(\infty) = \frac{1}{\lambda_2} \left( \frac{T_2 - T_1}{\frac{L_1}{\lambda_1} + \frac{L_2}{\lambda_2}} \right) (\alpha - (L_1 + L_2)) + T_2$

$$3) T_i = T_1(L_1) = \frac{1}{\lambda_1} \frac{T_2 - T_1}{\frac{L_1}{\lambda_1} + \frac{L_2}{\lambda_2}} L_1 + T_1$$

$$R = \frac{L}{\lambda S} \quad G = \frac{\lambda S}{L} \quad \begin{cases} G_1 = \frac{\lambda_1}{L_1} S \\ G_2 = \frac{\lambda_2}{L_2} S \end{cases}$$

$$T_i = \frac{L_1}{\lambda_1} \left( \frac{T_2 - T_1}{\frac{L_1 S_1}{\lambda_1 S_2} + \frac{L_2 S_2}{\lambda_2 S_2}} \right) = \frac{L_1}{\lambda_1 S_1} \frac{T_2 - T_1}{\frac{L_1}{\lambda_1 S_2} + \frac{L_2}{\lambda_2 S_2}}$$

$$= \frac{1}{G_1} \frac{T_2 - T_1}{\frac{1}{G_1} + \frac{1}{G_2}} + T_1$$

$$= \frac{(T_2 - T_1) G_2}{G_1 + G_2} + \frac{T_1 (G_1 + G_2)}{G_1 + G_2}$$

$$T_i = \frac{G_1 T_1 + G_2 T_2}{G_1 + G_2}$$

$$4) \dot{q}_1 = -\lambda_1 \frac{dT_1}{dx} \vec{e}_x$$

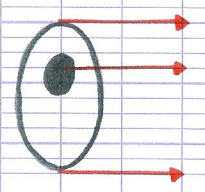
$$j = j_1 = j_2 = \frac{T_1 - T_2}{\frac{L_1}{\lambda_1} + \frac{L_2}{\lambda_2}}$$

$$j_1 = -\lambda_1 \left( \frac{1}{\lambda_1} \frac{T_2 - T_1}{\frac{L_1}{\lambda_1} + \frac{L_2}{\lambda_2}} \right)$$

$$j_2 = -\lambda_2 \left( \frac{1}{\lambda_2} \frac{T_2 - T_1}{\frac{L_1}{\lambda_1} + \frac{L_2}{\lambda_2}} \right)$$

$$q = \iint \vec{j} \cdot d\vec{S} = \iint j \vec{e}_x \cdot dS \vec{e}_x$$

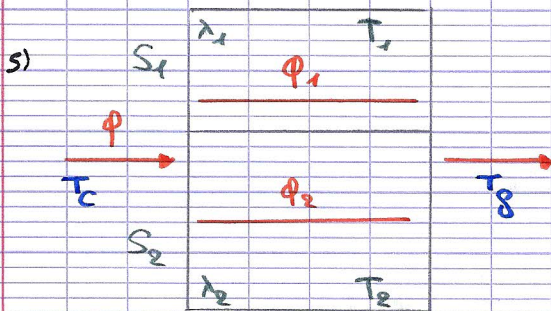
$$= j S$$



$$U = R_{eq} I$$

$$T_2 + T_1 = \left( \frac{L_1}{\lambda_1 S} + \frac{L_2}{\lambda_2 S} \right) \Phi$$

$$R_{eq} = \frac{L_1}{\lambda_1 S} + \frac{L_2}{\lambda_2 S} = R_1 + R_2$$



$$T_1(x) = c_1 x + b_1$$

$$T_2(x) = c_2 x + b_2$$

$$\begin{cases} T_1(x) = \frac{T_1 - T_c}{L} x + T_c \\ T_2(x) = \frac{T_2 - T_g}{L} x + T_g \end{cases}$$

$$\vec{j}_1 = -\lambda_1 \frac{dT_1}{dx} \vec{e}_x = \lambda_1 \left( \frac{T_c - T_1}{L} \right) \vec{e}_x$$

$$\vec{j}_2 = -\lambda_2 \frac{dT_2}{dx} \vec{e}_x = \lambda_2 \left( \frac{T_2 - T_g}{L} \right) \vec{e}_x$$

$$\Phi_1 = \iint \vec{j}_1 \cdot d\vec{S} = \frac{\lambda_1 S}{L} (T_c - T_1)$$

$$\Phi_2 = \frac{\lambda_2 S}{L} (T_2 - T_g)$$

$$\Phi_{tot} = \Phi_1 + \Phi_2$$

$$\Phi_{tot} = \left( \frac{\lambda_1 S}{L} + \frac{\lambda_2 S}{L} \right) (T_c - T_g)$$

$$U = RI \Rightarrow I = \frac{U}{R}$$

$$\Rightarrow \frac{1}{R_{eq}} = \frac{\lambda_1 S}{L} + \frac{\lambda_2 S}{L}$$

$$\Phi_{tot} = \frac{1}{R_{eq}} (T_c - T_g)$$

$$\Rightarrow \frac{1}{R_{eq}} = \frac{1}{R_1} + \frac{1}{R_2}$$

## Transfert conducto-convectif

Cas lorsque l'une des interfaces fait intervenir un fluide (liquide ou gaz)

Solide      fluide

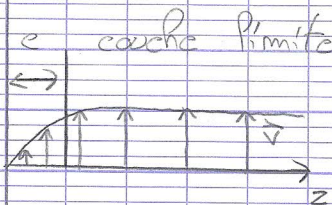
$\lambda_s$   
 $T_s$

$\lambda_f$   
 $T_f$

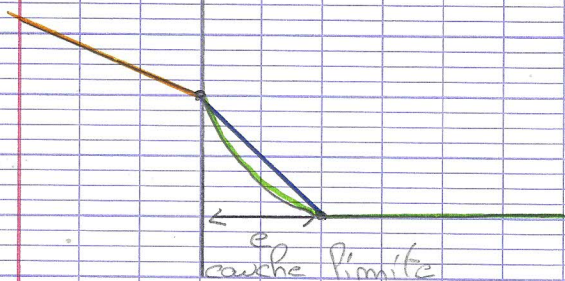
Continuité de  $\vec{j}$  à l'interface  
 $\vec{j}_s = -\lambda_s \frac{dT_s}{dz} \vec{e}_z$        $\vec{j}_f = -\lambda_f \frac{dT_f}{dz} \vec{e}_z$

$$\lambda_s \frac{dT_s}{dz} = \lambda_f \frac{dT_f}{dz}$$

Solide      fluide



Solide      fluide



$$\vec{j}_f = -\lambda_f \frac{T_f - T_s}{e} \vec{e}_z$$

$$\begin{aligned} \vec{j} &= -\frac{\lambda_f}{e} (T_{\text{fluide}} - T_{\text{paroi}}) \vec{e}_z \\ &= h (T_{\text{paroi}} - T_{\text{fluide}}) \vec{e}_z \end{aligned}$$

$$h = \frac{\lambda_f}{e} \quad \text{coef de transfert conducto-convectif.}$$

$T_s \lambda_s$

$T_{\text{paroi}}$

$T_{\text{fluide}}$

$T_s$

$T_{\text{paroi}}$

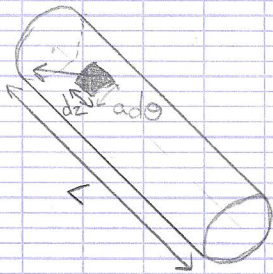
← coef h

$T_{\text{fluide}}$

Exercice 18: Isolation d'une canalisation d'eau chaude.

18.1

1)



$$\begin{aligned} \varphi &= \iint \vec{j} \cdot d\vec{S} \\ d\vec{S} &= a d\theta dz \vec{e}_r \\ \vec{j} &= h(T_1 - T_0) \vec{e}_r \\ \varphi &= \iint h(T_1 - T_0) a d\theta dz \\ \varphi &= h(T_1 - T_0) a \int_0^L dz \int_0^{2\pi} d\theta \\ \varphi_0 &= h(T_1 - T_0) 2\pi L a \end{aligned}$$

2)  $(T_1 - T_0) = \frac{1}{2\pi L a h} \varphi_0$

$$R_{Th} = \frac{1}{2\pi a L h} = \frac{1}{Sh}$$

18.2

1)  $\Delta T = 0$

$T(r, \theta, z)$

$\rightarrow T(r)$

2)  $\Delta g(r) = \frac{1}{r} \frac{d}{dr} \left( r \frac{dg}{dr} \right)$

$\Delta T = 0 \Rightarrow \frac{1}{r} \frac{d}{dr} \left( r \frac{dT}{dr} \right) = 0$

$\Rightarrow r \frac{dT}{dr} = A$

$\frac{dT(r)}{dr} = \frac{A}{r} \Rightarrow T(r) = A \ln r + B$

2 conditions limites

$$\begin{cases} T(r=a) = T_1 \\ T(r=b) = T_2 \end{cases} \Rightarrow \begin{cases} T_1 = A \ln a + B \\ T_2 = A \ln b + B \end{cases} \Rightarrow \begin{cases} A \ln b - A \ln a = T_2 - T_1 \\ A = \frac{T_2 - T_1}{\ln(b/a)} \end{cases}$$

$$T(r) = \frac{T_2 - T_1}{\rho_m(b/a)} \rho_m(r/a) + B$$

$$T(r) = \frac{T_2 - T_1}{\rho_m(b/a)} \rho_m(r/a) + T_1$$

$$3) \vec{j} = -\lambda \vec{\nabla} T = -\lambda \left( \frac{\partial T}{\partial r} \vec{e}_r + \frac{1}{r} \frac{\partial T}{\partial \theta} \vec{e}_\theta + \frac{\partial T}{\partial z} \vec{e}_z \right)$$

$$T(r) \Rightarrow \vec{j} = -\lambda \frac{dT(r)}{dr} \vec{e}_r$$

$$\frac{dT}{dr} = \frac{T_2 - T_1}{\rho_m(b/a)} = \frac{1}{r}$$

$$\vec{j}(r) = -\lambda \frac{T_2 - T_1}{\rho_m(b/a)} \frac{1}{r} \vec{e}_r$$

$$\vec{j}(a) = -\lambda \frac{T_2 - T_1}{\rho_m(b/a)} \frac{1}{a} \vec{e}_r$$

$$5) \varphi_1 = \lambda \frac{T_1 - T_2}{\rho_m(b/a)} 2\pi L$$

$$\varphi_2 = 2\pi L b h (T_2 - T_0)$$

$$6) T_1 - T_2 = \frac{\lambda (T_1 - T_2)}{bh \rho_m(b/a)} T_1 - T_0 + T_0 - T_2$$

$$T_1 - T_2 = \frac{T_1 - T_0}{1 + \frac{\lambda}{bh \rho_m(b/a)}}$$

$$8) \frac{\varphi_0}{\varphi_1} = \frac{a}{b} + \frac{ah}{\lambda} \rho_m\left(\frac{b}{a}\right)$$

$$9) \text{ si } h=0 \quad \frac{\varphi_0}{\varphi_1} = \frac{a}{b} < 1 \Rightarrow \varphi_0 < \varphi_1$$

$$10) \text{ si } h>0 \quad \frac{b}{a} \text{ grand } b \gg a \quad \frac{a}{b} = \varepsilon$$

$$\frac{\varphi_0}{\varphi_1} = \varepsilon + a \frac{h}{\lambda} \rho_m\left(\frac{1}{\varepsilon}\right)$$

$$1) \varphi = \iint \vec{j} \cdot d\vec{S}$$

$$\varphi(a) = \iint \vec{j}(a) \cdot d\vec{S}$$

$$= \iint \vec{j}(a) \cdot a d\theta dz \vec{e}_r$$

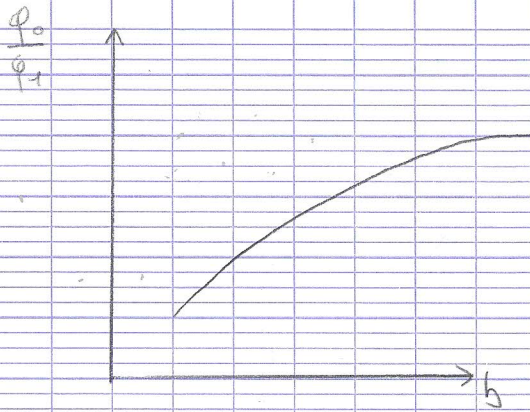
$$\varphi(a) = \int_0^{2\pi} \int_0^L \lambda \frac{T_1 - T_2}{\rho_m(b/a)} \frac{1}{a} a d\theta dz$$

$$\varphi_1 = \varphi(a) = \lambda \frac{T_1 - T_2}{\rho_m(b/a)} 2\pi L$$

1) pas d'accumulation de chaleur donc  $\varphi_1 = \varphi_2$

$$\varphi_1 = \text{flux en } r=a$$

$$r=b$$



$\frac{\Phi_0}{\Phi_1}$  augmente comme  $\ln\left(\frac{b}{a}\right)$  donc ~~cas~~  $\ln$  est  
 $\Rightarrow$  pas efficace.

en du coup, on prend  $e = b - a$  faible  
 $b = a + e \quad \frac{b}{a} = 1 + \frac{e}{a} \rightarrow e$

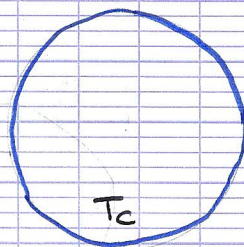
$$\frac{\Phi_0}{\Phi_1} = \frac{l}{1+e} + \frac{ah}{\lambda} \ln(1+e) \quad \text{if faut } \Phi_0/\Phi_1 > 1$$

$$= 1 - e + \frac{ah}{\lambda} e \quad \Rightarrow \frac{ah}{\lambda} > 1$$

$$= 1 + e\left(\frac{ah}{\lambda} - 1\right) \quad \text{if faut } \lambda < ah$$

$$h > \frac{\lambda}{a}$$

### Exercice 19



1)  $\Delta T = 0$  en st<sub>a</sub>

2)  $T(r, \theta, \varphi)$  invariance

de notation  $\theta, \varphi \Rightarrow T(r) + T = \text{cte}$  car bon conducteur

$$3) \text{ A } T(r) = 0 \Rightarrow \frac{1}{r^2} \frac{d}{dr} \left( r^2 \frac{dT}{dr} \right) = 0$$

$$\Rightarrow r^2 \frac{dT}{dr} = \text{cte} = A'$$

$$\Rightarrow \frac{dT}{dr} = \frac{A}{r} \Rightarrow T(r) = \frac{A}{r} + B$$

1) Conditions limites

$$T(r=a) = T_c$$

$$T(r \rightarrow +\infty) = T_g$$

$$\begin{cases} A/a + B = T_c \\ 0 + B = T_g \end{cases} \Rightarrow \begin{cases} A = T_c - T_g \\ B = T_g \end{cases}$$

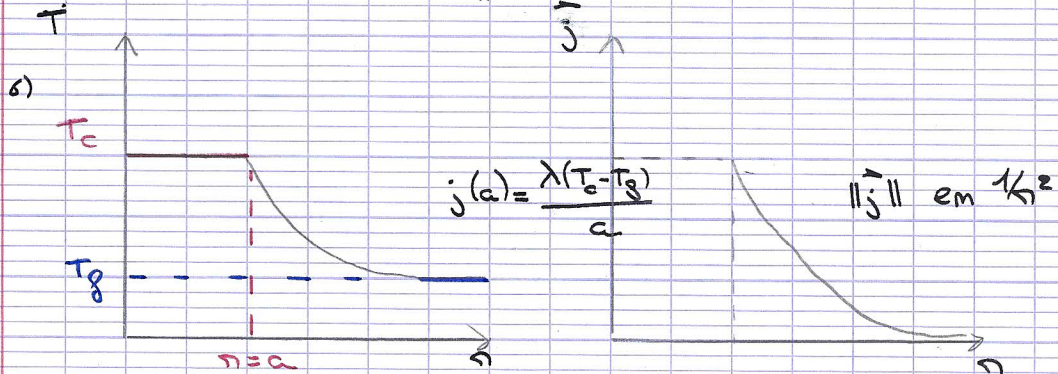
$$0 + B = T_g$$

$$\begin{cases} A = T_c - T_g \\ B = T_g \end{cases}$$

$$T(r) = (T_c - T_g) \frac{a}{r} + T_g$$

$$1) \vec{j} = -\lambda \vec{\nabla} T = -\lambda \frac{dT}{dr} \vec{e}_r = -\lambda (T_c - T_g) \times \left( \frac{-a}{r^2} \right) \vec{e}_r$$

$$\vec{j} = \lambda a (T_c - T_g) \frac{1}{r^2} \vec{e}_r \quad \text{j em } \frac{1}{r^2}$$



$$\Rightarrow \vec{\phi}_0 = \iint \vec{j} \, dS$$

$$\vec{j} = \lambda (T_c - T_g) a \frac{1}{r^2} \vec{e}_r$$

$$dV = r^2 \sin \theta \, d\theta \, d\varphi \, dr$$

$$\phi(r=a) = \iint \frac{\lambda a (T_c - T_g)}{a^2} a^2 \sin \theta \, d\theta \, d\varphi$$

$$\phi(r=a) = \lambda a (T_c - T_g) \left( \int_0^{2\pi} d\varphi \right) \times \left( \int_0^\pi \sin \theta \, d\theta \right)$$

$$= \lambda a (T_c - T_g) 4\pi$$

$$\phi_0 = 4\pi \lambda a (T_c - T_g)$$

$$q = j \cdot S$$

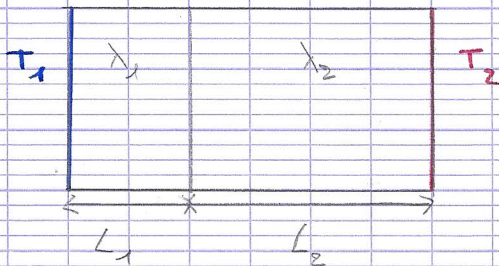
$$3) T_c - T_g = R_{th} \phi$$

$$R_{th} = \frac{1}{4\pi \lambda a}$$

$$s) R_{th} = \frac{1}{\pi \cdot 0,6 \cdot 10^{-3}} = 0,132 \cdot 10^8 \text{ W/K}$$

$$P_0 = \frac{T_c - T_g}{R_{Th}} \approx 5,3 \mu\text{W}$$

### Exercice 20



$$1) \vec{j} = -\lambda \vec{\nabla} T$$

$$\frac{dT}{dt} = K \Delta T$$

$\frac{dT}{dt} \rightarrow$  stat donc  $\Delta T = 0$

$$2) \vec{j}_1 = -\lambda_1 \frac{dT_1}{dx} \vec{e}_x$$

$$\frac{d^2 T(x)}{dx^2} = 0$$

$$\vec{j}_2 = -\lambda_2 \frac{dT_2}{dx} \vec{e}_x$$

$$\begin{cases} T_1(x) = a_1 x + b_1 \\ T_2(x) = a_2 x + b_2 \end{cases}$$

$$\Rightarrow \begin{cases} T_1(x) = a_1 x + T_0 \\ T_2(x) = a_2 x + T_0 \end{cases}$$

$$b_1 = b_2 = T_0$$

$\Downarrow$

Conditions limites

$$T_1(-L_1) = T_1$$

$$T_2(+L_2) = T_2$$

$$T_1(0) = T_0$$

$$T_2(0) = T_0$$

$$j_1(0) = j_2(0)$$

$$\begin{cases} T_1(x) = \frac{T_0 - T_1}{L_1} x + T_0 \\ T_2(x) = \frac{T_2 - T_0}{L_2} x + T_0 \end{cases}$$



$$3) \vec{j}_1 = -\lambda_1 \frac{dT_1}{dz} \vec{e}_x$$

$$\vec{j}_2 = -\lambda_2 \frac{T_0 - T_1}{L_2} \vec{e}_x \quad \text{idem} \quad \vec{j}_2 = -\lambda_2 \frac{T_2 - T_0}{L_2} \vec{e}_x$$

$$\Rightarrow T_0 = \frac{\lambda_1/L_1 T_1 + \lambda_2/L_2 T_2}{\lambda_1/L_1 + \lambda_2/L_2}$$

$$1) \left. \begin{aligned} G_1 &= \frac{\lambda_1}{L_1} = \frac{1}{R_1} \\ G_2 &= \frac{\lambda_2}{L_2} = \frac{1}{R_2} \end{aligned} \right\} T_0 = \frac{G_1 T_1 + G_2 T_2}{G_1 + G_2}$$

$$5) L_1 = L_2 \\ T_0 = \frac{\lambda_1 T_1 + \lambda_2 T_2}{\lambda_1 + \lambda_2}$$

6) con bois

$$\lambda_1 = \lambda_{\text{pau}} = 10 \text{ W} \cdot \text{m}^{-1} \cdot \text{K}^{-1}$$

$$\lambda_2 = \lambda_{\text{bois}} = 1 \text{ W} \cdot \text{m}^{-1} \cdot \text{K}^{-1}$$

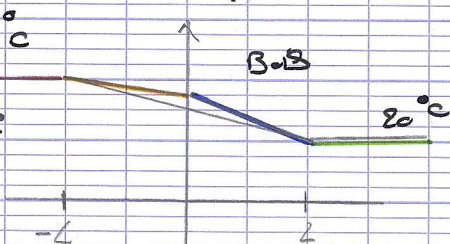
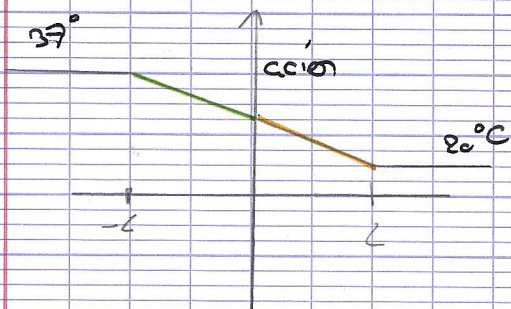
$$T_0^{\text{bois}} = \frac{10 \times 37 + 1 \times 20}{11} \approx 35,5^\circ \text{C}$$

con acción:

$$\lambda_1 = \lambda_{\text{pau}}$$

$$\lambda_2 = \lambda_{\text{acción}} = 100 \text{ W} \cdot \text{m}^{-1} \cdot \text{K}^{-1}$$

$$T_0 = \frac{10 \times 37 + 100 \times 20}{110} \approx 21,5^\circ \text{C}$$

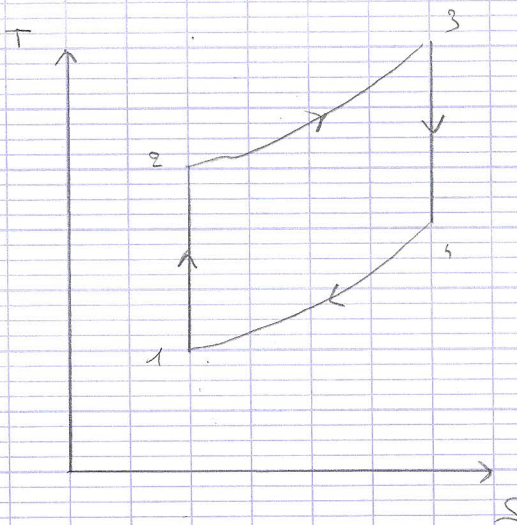
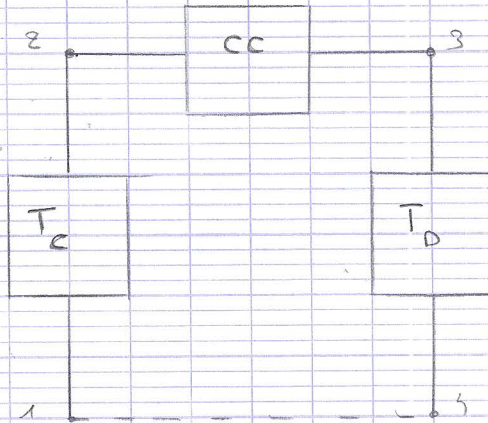


$$7) j = j_1 = j_2 \quad \text{em } X=0$$

$$j = \lambda_1 \frac{T_1 - T_0}{L}$$

$$j = \frac{\lambda_1}{L} \left( T_1 - \frac{T_1 - T_0}{L} \right)$$

Exercice 15



- 1 → 2 iso S
- 2 → 3 iso P réversible
- 3 → 4 iso P
- 4 → 1 iso P

1	2	3	4
P	1	6,5	6,5
T	300	T <sub>2</sub>	1300
S	0	0	s <sup>+</sup>

$$s_2 = s_1 = s_3^- = 0 \text{ J} \cdot \text{K}^{-1} \cdot \text{kg}^{-1}$$

$$s_3 = s_4 = s^+ = s_1 + c_p \ln \left( \frac{T_3}{T_1} \right) - \frac{R}{M} \ln \left( \frac{P_3}{P_1} \right)$$

$$= s_2 + c_p \ln \left( \frac{T_3}{T_2} \right) - \frac{R}{M} \ln \left( \frac{P_3}{P_2} \right)$$

$$\rightarrow s_3 = s_2 + c_p \ln \left( \frac{T_3}{T_2} \right) - \frac{R}{M} \ln \left( \frac{P_3}{P_2} \right)$$

$$s_3 = c_p \ln \left( \frac{T_3}{T_2} \right) = 931,5 \text{ J} \cdot \text{K}^{-1} \cdot \text{kg}^{-1}$$

2)  $\Delta(h + e_p + e_c) = w + q$

$\Delta h + \Delta e_p + \Delta e_c = w + q$       ici  $\Delta h = w + q$

1 → 2       $\Delta h_{12} = h_2 - h_1 = w_{12} + q_{12}$       → 0 car adiabatique

$\Rightarrow w_{12} = \Delta h_{12}$   
 $q_p \rightarrow T_1 \rightarrow T_2$

$w_{12} = c_p(T_2 - T_1) = 212 \text{ RJ} \cdot \text{kg}^{-1}$

$w_{31} = c_p(T_1 - T_3) = -535 \text{ RJ} \cdot \text{kg}^{-1}$

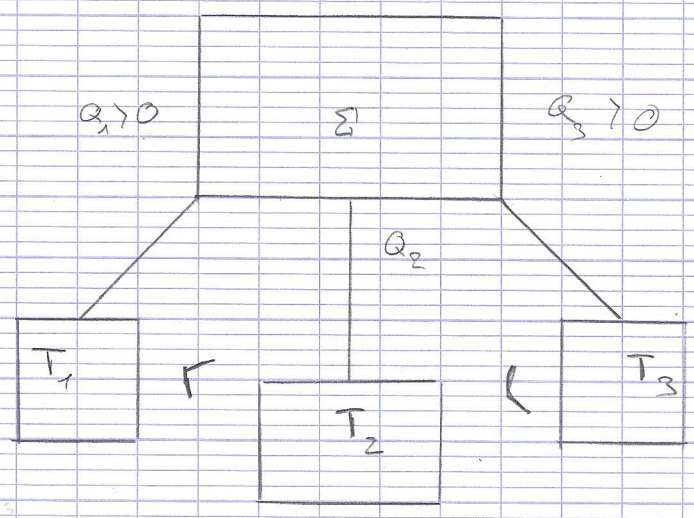
$w_{\text{tot}} = w_{12} + w_{31} = -323 \text{ RJ} \cdot \text{kg}^{-1}$   
 $< 0$  donc moteur.

3)  $\Delta h_{23} = w_{23} + q_{23}$   
 $\hookrightarrow q_{23} = c_p(T_3 - T_2) = 788 \text{ RJ} \cdot \text{kg}^{-1}$

4)  $e = \frac{w}{q_c} = \frac{-w_{\text{tot}}}{q_c} \approx 0,11$

~~$\Rightarrow w_{31} = c_p(T_1 - T_3) = -535 \text{ RJ}$   
 $w_{\text{tot}} = w_{12} + w_{31} = -323$~~

Exercice 13



$Q_1 > 0$  car but du frigo

2) cycle  $\Rightarrow \Delta U = 0 \Rightarrow w + Q_1 + Q_2 + Q_3 = 0$

$Q_1 + Q_2 + Q_3 = 0$

$\sum \frac{Q_i}{T_i} \leq 0 \Rightarrow \frac{Q_1}{T_1} + \frac{Q_2}{T_2} + \frac{Q_3}{T_3} \leq 0$

$$\Rightarrow -\frac{Q_3}{T_3} \geq \frac{Q_1}{T_1} + \frac{Q_2}{T_2} \quad \text{and} \quad Q_2 = Q_3 - Q_1$$

$$\Rightarrow -\frac{Q_3}{T_3} \geq \frac{Q_1}{T_1} - \frac{Q_3}{T_2} - \frac{Q_2}{T_2}$$

$$Q_3 \left( \frac{1}{T_2} - \frac{1}{T_3} \right) \geq Q_1 \left( \frac{1}{T_1} - \frac{1}{T_2} \right)$$

$$Q_3 \geq Q_1 \frac{\frac{1}{T_1} - \frac{1}{T_2}}{\frac{1}{T_2} - \frac{1}{T_3}}$$

$$3) \quad e = \frac{Q_2}{Q_3} = \frac{Q_1}{Q_3}$$

$$e \leq e_{\max} = \frac{\frac{1}{T_2} - \frac{1}{T_3}}{\frac{1}{T_1} - \frac{1}{T_2}}$$

$$1) \quad e_c = \frac{T_8}{T_c - T_8} = \frac{1/T_c}{1/T_8 - 1/T_c} = 18,5$$

$$\text{also } e_{\max} = \frac{1/T_c}{1/T_8 - 1/T_c} - \frac{1/T_3}{1/T_8 - 1/T_c} \leq e_c = 14,0$$

$$e_{\max} \leq e_c$$